

## BASIC AIR CONDITIONING FORMULAS

TO DETERMINE	EXPRESSED AS	COOLING	HEATING and/or HUMIDIFYING
Total Airflow	CFM <sub>T</sub>	1. $CFM_T = \frac{N_T V}{60 \text{ min./hr.}}$	1. $CFM_T = \frac{N_T V}{60 \text{ min./hr.}}$
Infiltration or Ventilation	CFM <sub>O</sub>	2. $CFM_O = \frac{N_O V}{60 \text{ min./hr.}}$	2. $CFM_O = \frac{N_O V}{60 \text{ min./hr.}}$
Number of Air Changes Per Hour – Total	N <sub>T</sub>	3. $N_T = \frac{CFM_T (60 \text{ min./hr.})}{V}$	3. $N_T = \frac{CFM_T (60 \text{ min./hr.})}{V}$
Number of Air Changes Per Hour – Outdoor Air	N <sub>O</sub>	4. $N_O = \frac{CFM_O (60 \text{ min./hr.})}{V}$	4. $N_O = \frac{CFM_O (60 \text{ min./hr.})}{V}$
Total Heat (H <sub>T</sub> )	Btuh	5. $H_T = CFM_T \times 4.5 \times (h_1 - h_2) = \text{Btuh}$	6. $H_T = CFM_T \times 4.5 \times (h_2 - h_1) = \text{Btuh}$
Sensible Heat (H <sub>S</sub> )	Btuh	7. $H_S = CFM_T \times 1.08 \times (T_1 - T_2) = \text{Btuh}$	8. $H_S = CFM_T \times 1.08 \times (T_2 - T_1) = \text{Btuh}$
Latent Heat (H <sub>L</sub> )	Btuh	9. $H_L = CFM_T \times .68 \times (W_1 - W_2) = \text{Btuh}$	10. $H_L = CFM_T \times .68 \times (W_2 - W_1) = \text{Btuh}$
Entering Air Temperature (T <sub>1</sub> ) (Mixed Air)	°F. D.B.	11. $T_1 = t_1 + \frac{CFM_O}{CFM_T} \times (t_2 - t_1) = \text{°F. D.B. } \textcircled{1}$ $\textcircled{1}$ If duct heat gain is a factor, add to T <sub>1</sub> : $\frac{\text{Duct Heat Gain (Btuh)}}{CFM_T \times 1.08}$	12. $T_1 = t_1 - \frac{CFM_O}{CFM_T} \times (t_1 - t_2) = \text{°F. D.B. } \textcircled{2}$ $\textcircled{2}$ If duct heat loss is a factor, subtract to T <sub>1</sub> : $\frac{\text{Duct Heat Loss (Btuh)}}{CFM_T \times 1.08}$
Leaving Air D.B. Temperature (T <sub>2</sub> )	°F. D.B.	13. $T_2 = T_1 - \frac{H_S}{CFM_T \times 1.08} = \text{°F. D.B.}$	14. $T_2 = T_1 + \frac{H_S}{CFM_T \times 1.08} = \text{°F. D.B.}$
Required Airflow	CFM <sub>T</sub>	15. $CFM_T = \frac{H_S \text{ (total)}}{1.08 \times (T_1 - T_2)} = \text{CFM}$ OR $CFM_T = \frac{H_S \text{ (internal)} \textcircled{3}}{1.08 \times (t_1 - T_2)} = \text{CFM}$ $\textcircled{3}$ Sensible load of outside air not included	16. $CFM_T = \frac{H_S}{1.08 \times (T_2 - T_1)} = \text{CFM}$
Enthalpy – Leaving Air (h <sub>2</sub> )	Btu/lb. dry air	17. $h_2 = h_1 - \frac{H_T}{CFM_T \times 4.5} = \text{Btu/lb. dry air}$	18. $h_2 = h_1 + \frac{H_T}{CFM_T \times 4.5} = \text{Btu/lb. dry air}$
Leaving Air W.B. Temperature	°F.W.B.	19. Refer to Enthalpy Table and read W.B. temperature corresponding to enthalpy of leaving air (h <sub>2</sub> ) (see #17).	20. Refer to Enthalpy Table and read W.B. temperature corresponding to enthalpy of leaving air (h <sub>2</sub> ) (see #18).
Heat Required to Evaporate Water Vapor Added to Ventilation Air	Btuh	21. $H_L = CFM_O \times .68 (W_3 - W_O) = \text{Btuh}$	22. $H_L = CFM_O \times .68 (W_3 - W_O) = \text{Btuh}$
Humidification Requirements	Lbs. water/hr.	23. $\left( \text{Make up Moisture} \right) = \frac{\text{Excess Latent Capacity of System} \times \% \text{ Run Time}}{1060 \text{ Btu/lb.}} = \text{Lbs./hr.}$ (Industrial Process Work)	24. $\left( \text{Make up Moisture} \right) = \frac{H_L \text{ loss Btuh (see #22)}}{1060 \text{ Btu/lb.}} = \text{Lbs./hr.}$

LEGEND	DERIVATION OF AIR CONSTANTS
<p> <b>CFM<sub>T</sub></b> = Total airflow cubic feet/min.  <b>CFM<sub>O</sub></b> = Outdoor air cubic feet/min.  <b>N<sub>T</sub></b> = Total air changes per hour  <b>N<sub>O</sub></b> = Outdoor air, air changes per hour  <b>V</b> = Volume of space cubic feet  <b>H<sub>T</sub></b> = Total heat Btuh  <b>H<sub>S</sub></b> = Sensible heat Btuh  <b>H<sub>L</sub></b> = Latent heat Btuh                      * <b>h<sub>1</sub></b> = Enthalpy or total heat of entering air                      * <b>h<sub>2</sub></b> = Enthalpy or total heat of leaving air  <b>T<sub>1</sub></b> = Temperature of entering air  <b>T<sub>2</sub></b> = Temperature of leaving air  <b>T<sub>adp</sub></b> = Apparatus dewpoint  <b>t<sub>1</sub></b> = Indoor design temperature  <b>t<sub>2</sub></b> = Outdoor design temperature  <b>W<sub>1</sub></b> = Grains of water/lb. of dry air at entering condition  <b>W<sub>2</sub></b> = Grains of water/lb. of dry air at leaving condition  <b>W<sub>3</sub></b> = Grains of water/lb. of dry air at indoor design conditions  <b>W<sub>O</sub></b> = Grains of water/lb. of dry air at outdoor design conditions                 </p>	<p>                     The air constants below apply specifically to standard air which is defined as dry air at 70°F and 14.7 P.S.I.A. (29.92 in. mercury column). They can, however, be used in most cooling calculations unless extremely precise results are desired.                 </p> <p>                     4.5 (To convert CFM to Lbs./hr.)                 </p> <p> <math>4.5 = \frac{60 \text{ Min./hr.}}{13.33} \text{ or } 60 \times .075</math> </p> <p>                     Where 13.33 is the specific volume of standard air (cu.ft./lb.) and .075 is the density (lbs./cu.ft.)                 </p> <p> <math>1.08 = \frac{.24 \times 60}{13.33} \text{ or } .24 \times 4.5</math> </p> <p>                     .24 BTU = specific heat of standard air (BTU/LB/°F)                 </p> <p> <math>.68 = \frac{60}{13.33} \times \frac{1060}{7000} \text{ or } 4.5 \times \frac{1060}{7000}</math> </p> <p>                     Where: 1060 = Average Latent Heat of water vapor. (BTU/LB.)                      7000 = Grains per lb.                 </p>

\* See Enthalpy of air (Total Heat Content of Air) Table for exact values.